

The following to be inserted for the paragraph beginning on pg. 4, line 17 and continuing to pg. 5, line 5:

B2

As used herein, fore and aft mode shake means front to back movement or displacement which is along the longitudinal axis of the vehicle and transverse of the axis of a mount and is identified by the letter F in Figure 2. Vertical mode shake means up and down movement or displacement which is along the axial axis of a mount and identified by the letter V in Figure 2. Lateral mode shake means side to side movement or displacement that is transverse of the longitudinal axis of the vehicle and is perpendicular to the fore and aft mode shake and identified by the letter L in Figure 3. Lateral shake stiffness is the displacement of a mount in the lateral mode shake direction divided by unit load. Vertical mode shake stiffness

The following to be inserted for the paragraph beginning on pg. 4, line 17 and continuing to pg. 5, line 5:

The following to be inserted for the paragraph on pg. 5, lines 6-9:

B3

An isolation mount according to the present invention is designated by the numeral 100 as shown in Figures 2-4. The mount 100 includes an elastomeric upper mount 10, an elastomeric lower mount 60 and a threaded fastener member 70.

The following to be inserted for the paragraph on pg. 5, lines 10-24:

B4

The upper mount 10 has a thimble member 26 with a flange portion 27 and an axially extending tubular portion 28. The mount 10 has a foamed elastomeric annular portion 14 which preferably has a top portion 14a with an inner diameter 16 and an outer diameter 18. The top portion has an axial length 15. Alternatively, the outer peripheral surface of the annular portion 14 may take any shape that can be used in the application, such as square, rectangular, polygonal, conical, triangular, elliptical or truncated conical or any other suitable shape. The annular portion 14 has an axially extending portion 20

B5 which is a close but sliding fit into the hole H in the subframe S. The axially extending portion 20 has an inner diameter 24 and a smaller outer diameter 22 than the outer diameter 18 of the annular portion 14. The axially extending portion 20 has an axial length 25 that extends the axial length L1 of the hole H in the subframe S. The bottom 15A of the annular portion 14 is placed adjacent the one side A of the subframe S. The axially extending tubular portion 28 is preferably elliptical in shape. Alternatively, the portion 28 may be round, parabolic with rounded ends or any shape suitable for practicing the invention.

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The following to be inserted for the paragraph on pg. 6, lines 1-5:

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B6 An insert 30 is disposed in the annular portion 14 and in the axially extending portion 20. The insert 30 has an outer diameter 32 and an inner peripheral shape 34. The peripheral shape 34 is preferably elliptical with a major diameter and a minor diameter which is similar to the shape of the annular axially extending layer 48.

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The following to be inserted for the paragraph on pg. 6, lines 6-22:

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B7 The insert 30 has an annular flange portion 40 and an axially extending tube portion 42. Alternatively, the peripheral shape 34 may be parabolic with rounded ends or any other suitable shape that could be used in practicing the invention. The outer diameter 32 of the insert is larger than the outer diameter 22 of the axially extending portion 20. The tube portion 42 also has an axially extending length 45 which is normally the same as the length L of the hole H in the subframe S for a purpose to be discussed later on. The thickness of the tube portion 42 may vary radially. The thickness is greatest adjacent to the axis of the minor diameter which corresponds to the direction of the lateral mode L and thinnest near the axis of the major diameter which corresponds to the fore and aft mode F. The insert 30 is disposed in the upper mount 12 such that it extends along the free end of the tube portion 42 but the flange portion 40 is disposed in the annular portion 14. Thus, there is a radial elastomeric layer portion 46 between the

B8

bottom surface 15A of the annular portion 14 and the bottom 41 of the flanged portion 40. Additionally, an annular axially extending elastomeric layer 48 is disposed against the elliptical inner peripheral shape 34 of the insert 30 and the tubular portion 28 of the thimble 26. The annular layer 48 has an elliptical shape.

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The following to be inserted for the paragraph on pg. 7, lines 5-19:

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B9

The elliptical shape of the layer 48 in the lateral displacement mode L provides an increased amount of volume of elastomer than a prior art round shaped layer, to absorb radial forces in the lateral mode direction L. This has two effects: 1) the large volume of elastomer provides the design engineer with another variable with which to tune the response of the mount to lateral displacement L to improve vehicle ride and handling characteristics; and 2) the larger volume of elastomer to absorb the lateral displacement forces to reduce stress in the elastomer. Thus, the elastomer has more area to absorb the force generated by the lateral displacement and as a result, the elastomer is exposed to a narrower range of material stress variations. Those skilled in the art will recognize that alternatively, the shape of the annular axially extending layer 48 and the tubular portion 28 of the thimble member and the peripheral shape 34 may be substantially parabolic in cross sectional area with rounded ends near the fore and aft, F, displacement mode or oblong in cross section or oval in cross sectional shape or any other non-circular shape suitable for practicing the invention.

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The following to be inserted for the paragraph beginning on pg. 7, line 20 and continuing to pg. 8, line 2:

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B10

The surface area of the tubular portion 28, in the lateral direction L, is preferably twice the surface area of the tubular portion 28 in the fore and aft direction F. Alternatively, the ration of surface area in the lateral direction L to surface area in the fore and aft direction F ranges from 1.05 to 4.0. The response of the mout to lateral forces is greater than the response to fore and aft forces, which is greater than the

B11 response to vertical forces. The vertical response ration is lower or softer than the lateral or force and aft response rates.

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The following to be inserted for the paragraph on pg. 8, lines 3-10:

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B12 The elastomeric lower mount 60 includes a lower annular portion 62. The annular portion 62 has an inner diameter 66 and an outer diameter of 64. A flat flanged annular member 68 is adjacent to the lower radial surface 65 of the lower annular portion 62. The inner diameter 66 is slightly greater than the outer diameter 22 of the axially extending portion 20. The lower mount has an axial length 63. The length 63 and the length 15 may be equal to each other. Optionally, the ratio of length 63 divided by length 15 may be greater than 1.0 or less than 1.0 depending on the vertical response rate desired.

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The following to be inserted for the paragraph beginning on pg. 8, lines 18-25:

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B13 The upper isolation member 21 and the lower isolator member 60 are made of foamed elastomeric materials. Preferably, the member 21, 60, respectively, are made of a foamed microcellular polyurethane material (MCU) which can be compression molded, cast or injection molded or processed by means wekk known in the prior art. Alternatively, the other foamed elastomers such as foamed fluorocarbon, foamed highly saturated nitrile, methyl acrylate acid polymer foam, silicone foam, EPDM foam, Neoprene® foam or the like and thermoplastic elastomers may be used in practicing the invention. Neoprene® is a registered trademark of DuPont.

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Page 9, Delete lines 11-16.